



为操纵面重心到操纵面铰链中心之间的距离。

操纵面铰链处的间隙非线性回复力矩  $\tilde{M}$  如图2所示。为间隙的大小。

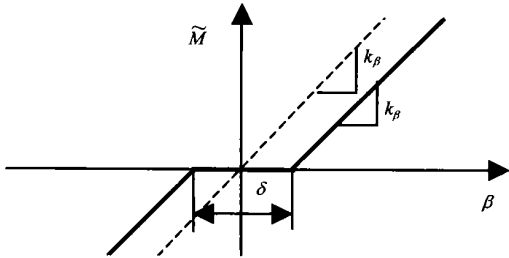


图2 间隙非线性

Fig.2 Freeplay nonlinearity

间隙非线性可表示为

$$\tilde{M} = \begin{cases} k(\beta + \delta/2) & \beta < -\delta/2 \\ 0 & -\delta/2 \leq \beta \leq \delta/2 \\ k(\beta - \delta/2) & \beta > \delta/2 \end{cases} \quad (1)$$

## 2 翼段的无量纲动力学方程

考虑间隙非线性在内的三自由度二维翼段的无量纲动力学方程为

$$M\ddot{q} + C\dot{q} + Kq + H = \frac{U^2}{\mu} C_e \quad (2)$$

其中:

$$M = \begin{bmatrix} 1 & \bar{x} & \bar{x} \\ \bar{x} & \bar{r}^2 & \bar{r}^2 + \bar{x}(c-a) \\ \bar{x} & \bar{r}^2 + \bar{x}(c-a) & \bar{r}^2 \end{bmatrix}$$

$$C = \text{diag} \left[ 2k_h \left( \frac{-b}{r} \right)^2, 2k_r \bar{r}^2, 2k_r \bar{r}^2 \left( \frac{-b}{r} \right)^2 \right]$$

$$K = \begin{cases} K_m = \text{diag} \left[ \left( \frac{-b}{r} \right)^2, \bar{r}^2, \bar{r}^2 \left( \frac{-b}{r} \right)^2 \right], & \beta < -\frac{\delta}{2} \\ K_n = \text{diag} \left[ \left( \frac{-b}{r} \right)^2, \bar{r}^2, 0 \right], & -\frac{\delta}{2} \leq \beta \leq \frac{\delta}{2} \\ K_m = \text{diag} \left[ \left( \frac{-b}{r} \right)^2, \bar{r}^2, \bar{r}^2 \left( \frac{-b}{r} \right)^2 \right], & \beta > \frac{\delta}{2} \end{cases}$$

$$H = \begin{cases} F_1 = \left[ 0 \ 0 \ \bar{r}^2 \left( \frac{-b}{r} \right)^2 \right]^T, & \beta < -\frac{\delta}{2} \\ F_2 = [0 \ 0 \ 0]^T, & -\frac{\delta}{2} \leq \beta \leq \frac{\delta}{2} \\ F_3 = \left[ 0 \ 0 \ \bar{r}^2 \left( \frac{-b}{r} \right)^2 \right]^T, & \beta > \frac{\delta}{2} \end{cases}$$

$$C_e = [-C_h(\beta) \ 2C(\beta) \ 2C(\beta)]^T$$

$$q = [h/b \ \beta]^T$$

其中:  $q$  为无量纲位移;  $\dot{q} = dq/dt$ ,  $t = t/U$  为无量纲时间;  $\mu = m/(U^2 b^2)$  为质量比;  $m$  为翼段的质量;  $\rho$  为空气密度;  $U = U/(U/b)$  为无量纲来

流速度;  $h, \beta$  分别为无耦合沉浮、俯仰和操纵面俯仰角频率;  $h, \beta$  分别为无耦合沉浮、俯仰和操纵面俯仰阻尼;  $C_h(\beta), C(\beta)$  和  $C(\beta)$  分别为升力系数、俯仰力矩系数和操纵面力矩系数;  $\bar{x} = x/b$  为翼段重心到弹性轴之间的无量纲距离;  $\bar{x} = x/b$  为操纵面重心到操纵面铰链中心之间的无量纲距离;  $\bar{r} = r/b$  为翼段绕弹性轴的无量纲回转半径;  $\bar{r} = r/b$  为操纵面绕操纵面铰链中心的无量纲回转半径。

## 3 非定常气动力的建立

引入气动力无量纲时间  $\tau = U t/b$ , 则气动力无量纲时间  $\tau$  与前面定义的  $t$  之间的关系为  $\tau = U t/(b) = U t$ 。根据 Theodorsen 理论并利用 Wagner 函数, 则  $C_h(\tau), C(\tau)$  和  $C(\tau)$  可表示为时域中的 Duhamel 积分的形式<sup>[3]</sup>

$$C_h(\tau) = C_h^1(\tau) + 2 \left\{ \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] + \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] \right\} \quad (3a)$$

$$C(\tau) = C^1(\tau) + C^2(\tau) + \left[ a + \frac{1}{2} \right] \left\{ \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] + \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] \right\} \quad (3b)$$

$$C(\tau) = C^1(\tau) + C^2(\tau) - \frac{F_{12}}{2} \left\{ \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] + \left[ q_s(0) \phi_W(U\tau) + \int_0^{\tau} \phi_W(U(\tau - \tau_0)) \frac{dq_s(\tau_0)}{d\tau_0} d\tau_0 \right] \right\} \quad (3c)$$

其中:

$$q_s(\tau) = \frac{1}{U} \left( \frac{\dot{h}}{b} \right) + \frac{1}{U} \left( \frac{1}{2} - a \right)$$

$$q_s(\tau) = \frac{F_{10}}{U} + \frac{F_{11}}{2U}$$

$$C_h^1(\tau) = \frac{F_1}{U} + \frac{F_2}{U^2} \left( \frac{\dot{h}}{b} \right) - \frac{a}{U^2} - \frac{F_4}{U} - \frac{F_1}{U^2}$$

$$C^1(\tau) = -\frac{1}{2U^2} \left( \frac{1}{8} + a^2 \right) + \frac{a}{2U^2} \left( \frac{\dot{h}}{b} \right) + \frac{F_7 + (c-a)F_1}{2U^2}$$

$$\begin{aligned}
 C^2(\cdot) &= -\frac{1}{2U} \left[ \frac{1}{2} - a \right] \cdot \left[ \begin{array}{c} F_4 - F_{11}/2 \\ -F_1 + F_8 + (c - a)F_4 - (1 - 2a)\frac{F_{11}}{4} \\ (F_4 - \frac{F_{12}}{2})\frac{F_{11}}{2} \end{array} \right] \\
 C^1(\cdot) &= -\frac{F_{13}}{U^2} \ddot{\cdot} + \frac{F_1}{2U^2} \left( \frac{\dot{h}}{b} \right) + \frac{F_3}{2U^2} \ddot{\cdot} \\
 C^2(\cdot) &= \frac{1}{2U} \left[ 2F_9 + F_1 - F_4 \left( a - \frac{1}{2} \right) \right] \cdot - \\
 &\quad \frac{1}{2} (F_5 - F_4 F_{10}) + \frac{1}{4U} F_4 F_{11} \cdot
 \end{aligned}$$

其中:  $\phi_W(\cdot)$  为 Wagner 函数;  $F_1 \sim F_{13}$  为 Theodorsen 常量。Wagner 函数可近似为

$$\phi_W(U(\cdot - 0)) = 1 - \frac{1}{2} e^{-1/2 U(\cdot - 0)} - \frac{1}{4} e^{-2U(\cdot - 0)} \quad (4)$$

其中:  $\beta_1 = 0.2048$ ;  $\beta_2 = 0.2952$ ;  $\beta_3 = 0.0557$ ;  $\beta_4 = 0.3333$ 。

定义气动力状态

$$z_1(\cdot) = \beta_1 \beta_1 U_0 \left( q_s(\cdot - 0) + q_s(\cdot - 0) \right) e^{-\beta_1 U(\cdot - 0)} d_0 \quad (5a)$$

$$z_2(\cdot) = \beta_2 \beta_2 U_0 \left( q_s(\cdot - 0) + q_s(\cdot - 0) \right) e^{-\beta_2 U(\cdot - 0)} d_0 \quad (5b)$$

由式(3)、式(4)、式(5)得

$$C_h(\cdot) = C_h^1(\cdot) + 2 \left( \beta_1 q_s(\cdot) + \beta_2 q_s(\cdot) \right) \cdot \phi_W(0) + z_1(\cdot) + z_2(\cdot) \quad (6a)$$

$$C(\cdot) = C^1(\cdot) + C^2(\cdot) + \left( \beta_1 + \frac{1}{2} \right) \cdot \left( \beta_1 q_s(\cdot) + \beta_2 q_s(\cdot) \right) \phi_W(0) + z_1(\cdot) + z_2(\cdot) \quad (6b)$$

$$C(\cdot) = C^1(\cdot) + C^2(\cdot) - \frac{F_{12}}{2} \left( \beta_1 q_s(\cdot) + \beta_2 q_s(\cdot) \right) \phi_W(0) + z_1(\cdot) + z_2(\cdot) \quad (6c)$$

由式(6)进一步得到

$$\frac{U^2}{\mu} C_e = \frac{1}{\mu} Z_1 \ddot{q} + \frac{U}{\mu} Z_2 \dot{q} + \frac{U^2}{\mu} Z_3 q + \frac{U^2}{\mu} Z_4 \quad (7)$$

其中:

$$Z_1 = \begin{bmatrix} - & a & F_1 \\ a & - & (1/8 + a^2) & F_7 + (c - a) F_1 \\ F_1 & - & 2 F_{13} & F_3/ \end{bmatrix}$$

$$Z_2 = \begin{bmatrix} - & - & (\frac{3}{2} - a) \\ (a + \frac{1}{2}) & - & (a - \frac{1}{2})^2 \\ - & \frac{F_{12}}{2} & 2 F_9 + F_1 + (a - \frac{1}{2}) (\frac{F_{12}}{2} - F_4) \end{bmatrix}$$

$$Z_3 = \begin{bmatrix} 0 & - & - & F_{10} \\ 0 & (a + \frac{1}{2}) & (a - \frac{1}{2}) & F_{10} - F_4 \\ 0 & - & \frac{F_{12}}{2} & \frac{(-2F_5 + 2F_4 F_{10} - F_{12} F_{10})}{2} \end{bmatrix}$$

$$Z_4 = \begin{bmatrix} - & 2 & - & 2 \\ (2a + 1) & (2a + 1) & & \\ - & F_{12} & - & F_{12} \end{bmatrix}, \quad J = [ \beta_1 \quad \beta_2 ]^T$$

气动力状态满足微分方程

$$\dot{z} = U^2 G_1 z + G_2 \dot{q} + U G_3 q \quad (8)$$

其中:

$$G_1 = \begin{bmatrix} - & \beta_1 & \beta_1^2 \\ & - & \beta_2 & \beta_2^2 \end{bmatrix}$$

$$G_2 = \begin{bmatrix} \beta_1 \beta_1 & \beta_1 \beta_1 (1/2 - a) & \beta_1 \beta_1 F_{11}/(2) \\ \beta_2 \beta_2 & \beta_2 \beta_2 (1/2 - a) & \beta_2 \beta_2 F_{11}/(2) \end{bmatrix}$$

$$G_3 = \begin{bmatrix} 0 & \beta_1 \beta_1 & \beta_1 \beta_1 F_{10}/ \\ 0 & \beta_2 \beta_2 & \beta_2 \beta_2 F_{10}/ \end{bmatrix}$$

引入状态变量

$$X = [ x_1 \quad x_2 \quad x_3 \quad x_4 \quad x_5 \quad x_6 \quad x_7 \quad x_8 ]^T = [ h/b \quad \dot{h}/b \quad \cdot \quad \cdot \quad \beta_1 \quad \beta_2 ]^T \quad (9)$$

由式(2)、式(7)、式(8)得到状态空间中翼段的无量纲分段线性动力学方程为

$$\dot{X} = A(U) X - F_d^1, R_1 = \left\{ \begin{array}{l} X \quad \mathbf{R}^8 / x_3 < - \frac{1}{2} \end{array} \right\} \quad (10a)$$

$$\dot{X} = B(U) X, R_2 = \left\{ \begin{array}{l} X \quad \mathbf{R}^8 / - \frac{1}{2} \quad x_3 \quad \frac{1}{2} \end{array} \right\} \quad (10b)$$

$$\dot{X} = A(U) X + F_d^1, R_3 = \left\{ \begin{array}{l} X \quad \mathbf{R}^8 / x_3 > \frac{1}{2} \end{array} \right\} \quad (10c)$$

其中:

$$A(U) = \begin{bmatrix} \mathbf{0}_{3 \times 3} & \mathbf{I}_{3 \times 3} & \mathbf{0}_{3 \times 2} \\ - & \tilde{M}^{-1} \bar{K}_m & - & \tilde{M}^{-1} \bar{C} & - & \tilde{M}^{-1} \bar{D} \\ U G_3 & & G_2 & & U^2 G_1 \end{bmatrix}$$

$$B(U) = \begin{bmatrix} \mathbf{0}_{3 \times 3} & \mathbf{I}_{3 \times 3} & \mathbf{0}_{3 \times 2} \\ - & \tilde{M}^{-1} \bar{K}_n & - & \tilde{M}^{-1} \bar{C} & - & \tilde{M}^{-1} \bar{D} \\ U G_3 & & G_2 & & U^2 G_1 \end{bmatrix}$$

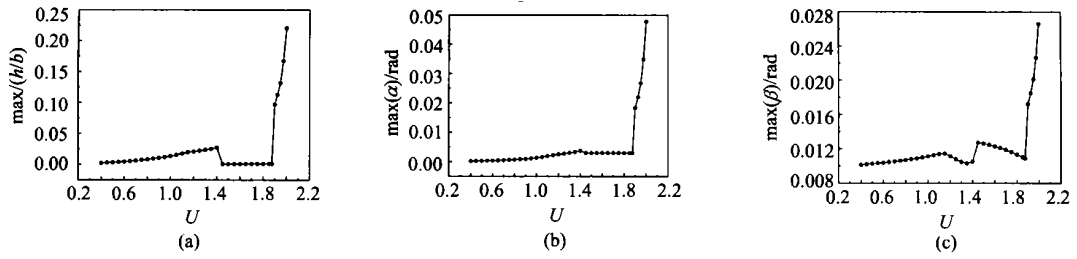


图3 极限环响应的幅值

Fig. 3 The amplitude of limit cycle response

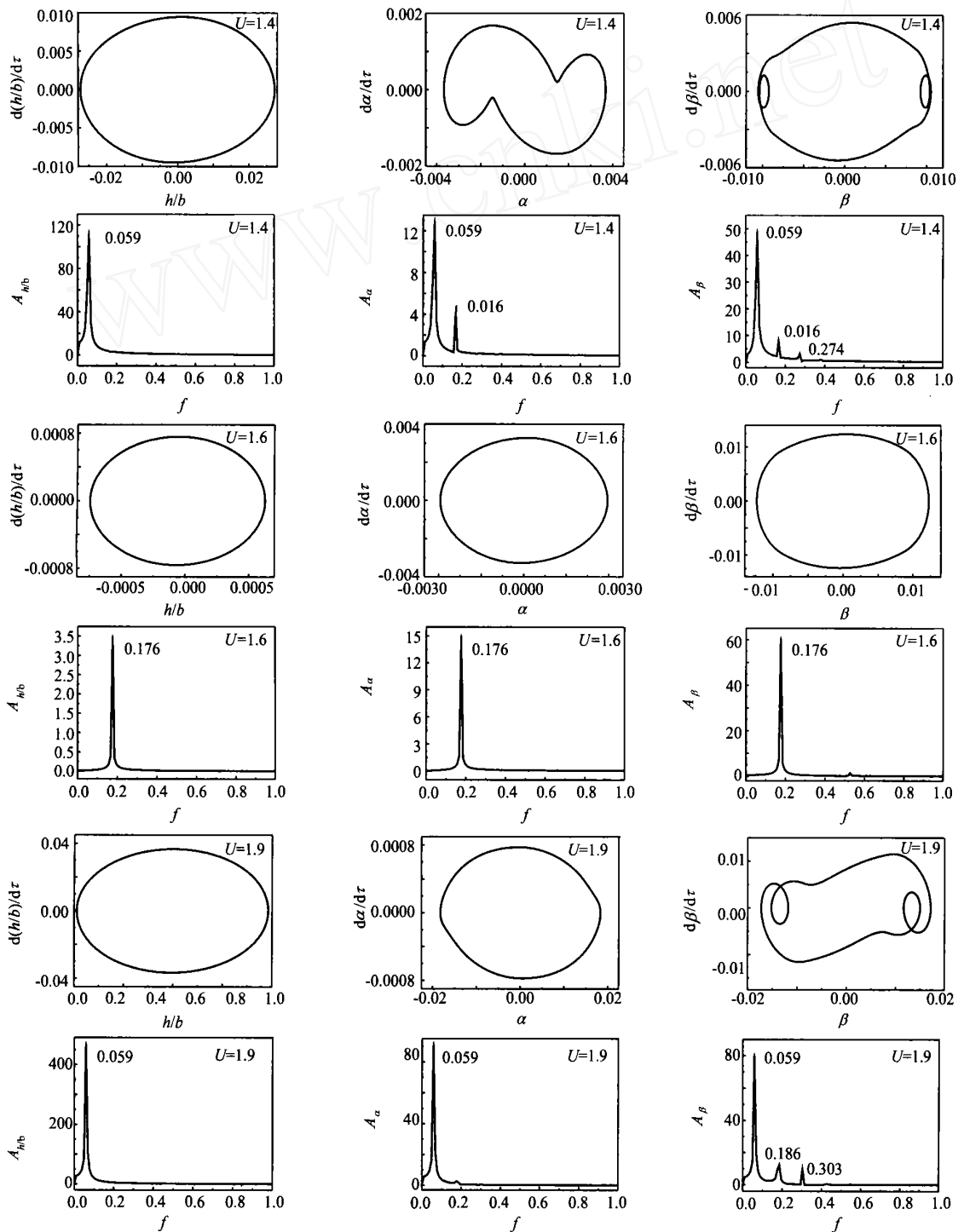


图4 不同  $U$  下极限环的相图和频谱

Fig. 4 The phase diagrams and spectrum of limit cycle oscillations for various  $U$

$$F_d^1 = \begin{bmatrix} \mathbf{0}_{3 \times 1} \\ \tilde{M}^{-1} F^1 \\ \mathbf{0}_{2 \times 1} \end{bmatrix}, \quad \tilde{M} = M - \frac{1}{\mu} Z_1$$

$$\bar{K}_m = K_m - \frac{U^2}{\mu} Z_3, \quad \bar{K}_n = K_n - \frac{U^2}{\mu} Z_3$$

$$\bar{C} = C - \frac{U}{\mu} Z_2, \quad \bar{D} = \frac{U^2}{\mu} Z_4$$

#### 4 数值结果

气动弹性响应分析所用到的仿真参数如下:

$a = -0.2$ ,  $c = 0.5$ ,  $\bar{x} = 0.2$ ,  $\bar{x} = 0.008$ ,  $\bar{r} = 0.5$ ,  $\bar{r} = 0.06$ ,  $\mu = 30$ ,  $h/a = 0.3$ ,  $\gamma = 1.5$ ,  $h = 0.016$ ,  $\delta = 0.006$ ,  $\epsilon = 0.004$ ,  $\theta = 1^\circ$

沉浮运动  $h/b$ , 俯仰运动  $\alpha$  和操纵面俯仰运动  $\delta$  的极限环响应幅值和无量纲来流速度的关系如图3所示。翼段运动方程(10b)是导致系统出现极限环振动的原因。通过分析此方程的Jacobi矩阵的特征值,得知系统的Hopf分岔点出现在  $U = 0.37$  处,此时在零点附近分化出极限环。结果表明系统的3个自由度在  $U = 1.4$  和  $U = 1.9$  处的响应幅值都存在跳跃现象。当  $U > 1.9$  时,极限环振动的幅值随无量纲来流速度的增加急速增长;当不考虑间隙非线性时,通过根轨迹分析可知系统的线性颤振速度为  $U = 2.0$ ,所以当  $U > 2.0$  时,系统表现为发散运动。这与文献[3]指出的间隙非线性不改变系统的颤振速度的结论是一致的。在本文给定的参数下,由间隙非线性导致的极限环振动表现为单谐波极限环和多谐波极限环两种形式,但均表现为周期-1运动(见图4)。对不同  $U$ ,翼段沉浮运动均表现为单谐波极限环。在极限环振动幅值出现跳跃前( $U = 1.4$ ),翼段俯仰运动  $\alpha$  和操纵面俯仰运动  $\delta$  均表现为多谐波极限环振动;幅值出现跳跃后( $U = 1.6$ ),系统3个自由度的响应均表现为单谐波极限环振动;图4还展示了系统极限环响应的幅值出现第2次跳跃之前( $U = 1.9$ )的极限环响应的相轨迹和频谱。

图5表明系统极限环振动的无量纲基频在

$U = 1.4$  和  $U = 1.9$  处也分别出现一次由低频到高频和由高频到低频的较大的跳跃。

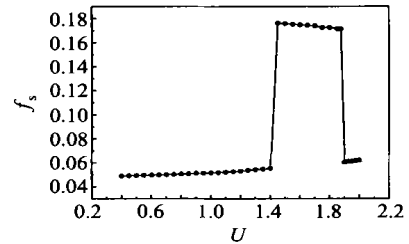


图5 基频  $f_s$  与流速的关系

Fig. 5 The relationship between fundamental frequency  $f_s$  and air speed  $U$

#### 参考文献

- [1] Price S J. The aeroelastic response of a two-dimensional airfoil with bilinear and cubic structural nonlinearities[J]. Journal of Fluids and Structures, 1995, 9(2): 175 - 193.
- [2] Liu L, Wong Y S. Nonlinear aeroelastic analysis using the point transformation method, Part I: freeplay models [J]. Journal of Sound and Vibration, 2002, 253(2): 447 - 469.
- [3] Nagarajan H, Gordon J, et al. Unsteady aerodynamics of a flapped airfoil in subsonic flow by indicial concepts[R]. AIAA paper 1228, 1995.

作者简介:



赵永辉(1971-) 男,黑龙江克东人,博士后,毕业于哈尔滨工业大学航天工程与力学系一般力学专业,主要从事非线性动力学分析、气动弹性力学分析与控制和模态参数识别等方面的研究工作。Email: hyhu@nuaa.edu.cn, 联系电话:(025) 4891672



胡海岩(1956-) 男,上海人,博士,教授,博士生导师。长期从事非线性动力学和振动控制研究。发表论文116篇,其中38篇被SCI和EI收录,他人引用300余次。出版著作《Dynamics of Controlled Mechanical Systems with Delayed Feedback》等3部。E-mail: hhyae@nuaa.edu.cn, 联系电话:(025) 4895000

(责任编辑:李铁柏)